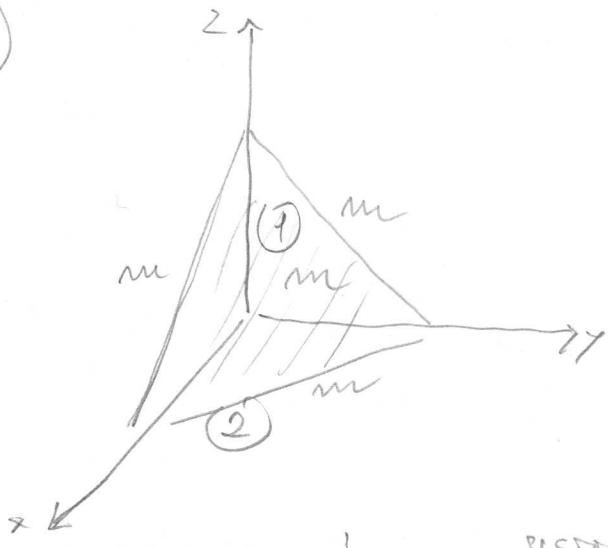


1)



- a) matrice di inerzia  
b) sistemi, momenti e lesi  
principali di inerzia

c)

$$I_{zz}^{(1)} = \int_m^l x^2 dx = \frac{l}{2} \int_0^l x^2 dx = \frac{l}{6} \frac{1}{3} ml^2 = \frac{1}{3} ml^2$$

$$I_{xx}^{(1)} = I_{yy}^{(1)}$$

$$\begin{aligned} I_{xy}^{(1)} &= -\frac{m}{l^2/2} \int_0^l x dx \int_0^{l-x} z dy = \\ &= -\frac{2ml}{l^2} \int_0^l x dx \left(\frac{l-x}{2}\right)^2 = \\ &= -\frac{2ml}{l^2} \int_0^l dx \left(l^2x - 2x^2l + x^3\right) = -\frac{ml}{l^2} \left(\frac{l^4}{2} - \frac{2}{3}l^4 + \frac{l^4}{4}\right) \\ &= -\frac{1}{12} ml^2 = I_{xz}^{(1)} = I_{zy}^{(1)} \end{aligned}$$

$$I_{zz}^{(1)} = \frac{1}{12} ml^2 \begin{pmatrix} 4 & -1 & -1 \\ -1 & 4 & -1 \\ -1 & -1 & 4 \end{pmatrix}$$

$$I_{yy}^{(1)} = \frac{1}{3} ml^2 = I_{xx}^{(1)}$$

$$I_{xz}^{(1)} = I_{yz}^{(1)} = 0$$

$$\begin{aligned} I_{xy}^{(2)} &= -\frac{m}{l} \int_0^l x(l-x) dx = -\frac{m}{l} \left(\frac{l x^2}{2} - \frac{x^3}{3}\right) \Big|_0^l \\ &= -\frac{m}{l} \left(\frac{l^3}{2} - \frac{l^3}{3}\right) = -\frac{1}{6} ml^2 \end{aligned}$$

$$I_{zz}^{(2)} = I_{xx}^{(2)} + I_{yy}^{(2)} = \frac{2}{3} ml^2$$

$$I = \frac{1}{12} ml^2 \begin{pmatrix} 4 & -2 & 0 \\ -2 & 4 & 0 \\ 0 & 0 & 2 \end{pmatrix}$$

$$I^{ASSE} = \frac{1}{12} ml^2 \begin{pmatrix} 16 & -2 & -2 \\ -2 & 16 & -2 \\ -2 & -2 & 16 \end{pmatrix}$$

$$I = \frac{1}{12} ml^2 \begin{pmatrix} 20 & -3 & -3 \\ -3 & 20 & -3 \\ -3 & -3 & 20 \end{pmatrix}$$

ci sono tre assi di simmetria a  $120^\circ$  l'uno dall'altro  
che si intersecano nella base incidenza al

$$\bar{\omega}_1 = \begin{pmatrix} 1 \\ 1 \\ 1 \end{pmatrix}$$

che quindi è parallelo all'asse di rotazione.

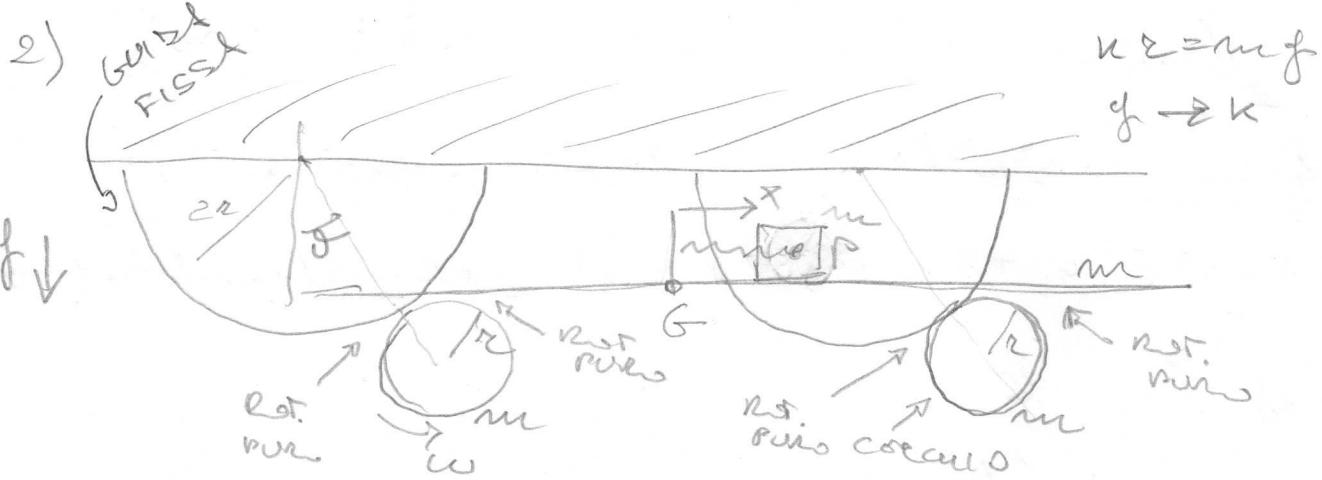
$$I \bar{\omega}_1 = I_1 \bar{\omega}_1 \quad \text{con} \quad I_1 = \frac{7}{6} ml^2 \quad (\lambda_1 = 14)$$

$$\lambda_1 + \lambda_2 + \lambda_3 = \text{Tr} \begin{pmatrix} 20 & -3 & -3 \\ -3 & 20 & -3 \\ -3 & -3 & 20 \end{pmatrix} = 60$$

$$\text{e } \lambda_2 = \lambda_3 \Rightarrow \lambda_2 = \lambda_3 = 23$$

$$\Rightarrow I_2 = I_3 = \frac{23}{12} ml^2$$

$\bar{\omega}_2$  e  $\bar{\omega}_3$  sono ora ortogonali rispetto ai vettori  
ortogonali intorno a  $\bar{\omega}_1$ .



$$V = \frac{1}{2} kx^2 + 4mg(-3r \cos \theta)$$

a)  $V$ , point stat.  
 b)  $T$   
 c) accel. excent.

$$\frac{\partial V}{\partial x} = kx$$

$$\frac{\partial V}{\partial \theta} = 12mr^2 \sin \theta$$

$\Rightarrow$  point stat.  $\begin{cases} x = 0 \\ \theta = 0 \end{cases}$

$$\frac{\partial^2 V}{\partial x^2} = k, \quad \frac{\partial^2 V}{\partial x \partial \theta} = 0, \quad \frac{\partial^2 V}{\partial \theta^2} = 12mr^2 \cos \theta$$

$$\mathbf{B} = \begin{pmatrix} k & 0 \\ 0 & 12mr^2 \end{pmatrix}$$

$$\omega = 3\dot{\theta}$$

$$\mathbf{f} = \left( 3r \sin \theta - 3r\dot{\theta} + \text{const.}, -3r \cos \theta + \text{const} \right)$$

$$\mathbf{P} = \mathbf{f} + (\mathbf{x}, \dot{\theta})$$

$$\dot{\mathbf{f}} = (3r \cos \theta - 3r\ddot{\theta}, 3r \sin \theta \dot{\theta})$$

$$\dot{\mathbf{P}} = (3r \cos \theta - 3r\ddot{\theta} + \ddot{x}, 3r \sin \theta \dot{\theta})$$

$$\dot{\mathbf{f}}^2 = 18r^2\dot{\theta}^2 - 18r^2\dot{\theta}^2 \cos \theta = 18r^2\dot{\theta}^2(1 - \cos \theta)$$

$$\dot{\mathbf{P}}^2 = 18r^2\dot{\theta}^2(1 - \cos \theta) + \dot{x}^2 + 6r\dot{\theta}\dot{x}(\cos \theta - 1)$$

$$\begin{aligned}
 T &= 2 \frac{1}{2} (mr^2 + mr^2) (\dot{\theta})^2 + \frac{1}{2} m [18r^2\dot{\theta}^2(1-\cos\theta) \\
 &\quad + \frac{1}{2} m \{ 18r^2\dot{\theta}^2(1-\cos\theta) - 6r\dot{\theta}\dot{x}(1-\cos\theta) + \ddot{x}^2 \}] \\
 &= 18mr^2\dot{\theta}^2 + 18mr^2\dot{\theta}^2(1-\cos\theta) - 3r\dot{\theta}\dot{x}m(1-\cos\theta) + \frac{1}{2} m \ddot{x}^2
 \end{aligned}$$

$$A = m \begin{pmatrix} 1 & 0 \\ 0 & 3Gr^2 \end{pmatrix}$$

$$\omega_1 = \sqrt{\frac{k}{m}}, \quad \omega_2 = \sqrt{\frac{k}{3m}}$$

Le mosse di oscillazione di  $x$  e  $\theta$  sono attaccate.